

Technical Information



Stainless Steel Springs

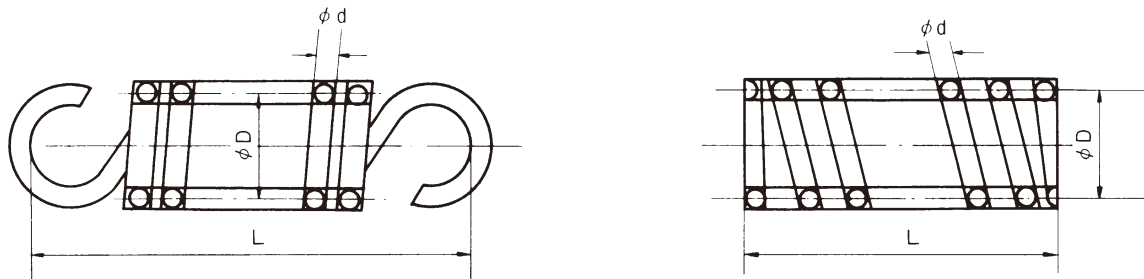
We have corrected the existing defects of surface treatments given to springs made from steel or alloyed metal, as well as other various mechanistic faults. As a result, we are proud to provide products that have earned customers' trust and praise. We hope that you will also appreciate our superior products and technology.

Type of Material	SUS304	SUS631	Piano Wire	Nickel Silver Wire	Phosphor Bronze Wire
Elastic Coefficient					
Shear Modulus (G)	73,550N/mm ²	76,492N/mm ²	78,453N/mm ²	38,246N/mm ²	41,188N/mm ²
Young's Modulus (E)	186,320N/mm ²	196,133N/mm ²	205,940N/mm ²	101,989N/mm ²	107,873N/mm ²

Mechanical Properties of Age-Hardened High-Tension Stainless Steel SUS631 - Precipitation Hardening

HRC	Pre-Hardening Mechanical Properties					Post-Hardening Mechanical Properties			
	Treatment	Tensile Test		Bend Test		Tensile Test		Flexural Strength Test	Hardness
		Tensile Strength N/mm ²	Elongation %	Bend Angle	Inside Radius	Tensile Strength N/mm ²	Elongation %	Threshold Limit Kb N/mm ²	HRC
22	R.H	892 - 1,030	8 - 13	180°	2 × Material Thickness	1,481	6 - 12	588	47
38	C.H	1,206 or over	6 - 10	180°		1,589	5 - 9	834	51

General Formula for Spring Design



d: Wire diameter D: Mean diameter N: Number of active coils W: Axial load δ: Deflection
G: Shear modulus (Stainless Steel Wire = approx. 70,000N/mm²)

Finding...				
			b ≤ 2c	b > 2c
Deflection from load	$\delta = \frac{8WND^3}{Gd^4}$	$\delta = \frac{5.6WND^3}{Gb^4}$	$\delta = \frac{2.79WND^3(b^2+c^2)}{Gb^3c^3}$	$\delta = \frac{2.35WND^3}{Gc^3(b-0.63c)}$
Deflection from shear stress	$\delta = \frac{\pi fsND^2}{Gd}$	$\delta = \frac{2.35fsND^2}{Gb}$	$\delta = \frac{3.5fsND^2(b^2+c^2)}{Gbc(2b+c)}$	$\delta = \frac{2.9fsND^2b^2}{Gc(2b+c)(b-0.63c)}$
Shear stress from load	$fs = \frac{8WD}{\pi d^3}$	$fs = \frac{2.38WD}{b^3}$	$fs = \frac{0.8WD(2b+c)}{b^2c^2}$	$fs = \frac{0.8WD(2b+c)}{b^2c^2}$
Shear stress from deflection	$fs = \frac{\delta Gd}{\pi ND^2}$	$fs = \frac{0.425\delta Gb}{ND^2}$	$fs = \frac{0.28\delta G(2b^2c+bc^2)}{ND^2(b^2+c^2)}$	$fs = \frac{0.34\delta Gc(2b+c)(b-0.63c)}{ND^2b^2}$
Load from estimated shear stress	$W = \frac{fs\pi d^3}{8D}$	$W = \frac{0.42fsb^3}{D}$	$W = \frac{1.25fsb^2c^2}{D(2b+c)}$	$W = \frac{1.23fsb^2c^2}{D(2b+c)}$
Load from deflection	$W = \frac{\delta d^4 G}{8ND^3}$	$W = \frac{\delta Gb^4}{5.6ND^3}$	$W = \frac{\delta Gb^3c^3}{2.79ND^3(b^2+c^2)}$	$W = \frac{\delta Gc^3(b-0.63c)}{2.35ND^3}$
The number of active coils	$N = \frac{\delta Gd^4}{8WD^3}$	$N = \frac{\delta Gb^4}{5.6WD^3}$	$N = \frac{\delta Gb^3c^3}{2.79WD^3(b^2+c^2)}$	$N = \frac{\delta Gc^3(b-0.63c)}{2.35WD^3}$

With respect to tension springs, the initial tension is set to 0.

Initial tension calculations

$$\delta = \frac{\pi(fs-fs_0)ND^2}{Gd} \quad P_0: \text{Initial tension} \quad fs_0: \text{Residual stress} \left(\frac{G}{100c}\right) \quad c: \text{Spring index} \left(\frac{D}{d}\right)$$

$$P_0 = \frac{\pi fs_0 d^3}{8D} \quad fs = \frac{\delta Gd}{\pi ND^2} + fs_0$$